

WL-TM-97-3016



HEAT TRANSFER IN THIN LIQUID FILMS

Rama S. R. Gorla
CSA ENGINEERING
2850 W. BAYSHORE ROAD
APLO ALTO CA 94303-3843

Larry W. Byrd
THERMAL STRUCTURES BRANCH
STRUCTURES DIVISION
FLIGHT DYNAMICS DIRECTORATE
WRIGHT LABORATORY

SEPTEMBER 1996

FINAL REPORT FOR PERIOD 05/31/96 – 08/31/96

Approved for public release; distribution unlimited

19970416 016

FLIGHT QUALITY INSPECTED

FLIGHT DYNAMICS DIRECTORATE
WRIGHT LABORATORY
AIR FORCE MATERIEL COMMAND
WRIGHT-PATTERSON AIR FORCE BASE, OH 45433-7552

NOTICE

USING GOVERNMENT DRAWINGS, SPECIFICATIONS, OR OTHER DATA INCLUDED IN THIS DOCUMENT FOR ANY PURPOSE OTHER THAN GOVERNMENT PROCUREMENT DOES NOT IN ANY WAY OBLIGATE THE US GOVERNMENT. THE FACT THAT THE GOVERNMENT FORMULATED OR SUPPLIED THE DRAWINGS, SPECIFICATIONS, OR OTHER DATA DOES NOT LICENSE THE HOLDER OR ANY OTHER PERSON OR CORPORATION; OR CONVEY ANY RIGHTS OR PERMISSION TO MANUFACTURE, USE, OR SELL ANY PATENTED INVENTION THAT MAY RELATE TO THEM.

THIS REPORT IS RELEASABLE TO THE NATIONAL TECHNICAL INFORMATION SERVICE (NTIS). AT NTIS, IT WILL BE AVAILABLE TO THE GENERAL PUBLIC, INCLUDING FOREIGN NATIONS.

THIS TECHNICAL REPORT HAS BEEN REVIEWED AND IS APPROVED FOR PUBLICATION.



LARRY W. BYRD
Mechanical Engineer
Thermal Structures Section
Structural Integrity Branch



CHRISTOPHER L. CLAY
Technical Manager
Thermal Structures Section
Structural Integrity Branch



JEROME PEARSON, Chief
Structural Integrity Branch
Structures Division

IF YOUR ADDRESS HAS CHANGED, IF YOU WISH TO BE REMOVED FROM OUR MAILING LIST, OR IF THE ADDRESSEE IS NO LONGER EMPLOYED BY YOUR ORGANIZATION PLEASE NOTIFY WL/FIBEB, BLDG 45, 2130 EIGHTH ST STE 1, WRIGHT-PATTERSON AFB OH 45433-7542 TO HELP MAINTAIN A CURRENT MAILING LIST.

Do not return copies of this report unless contractual obligations or notice on a specific document requires its return.

REPORT DOCUMENTATION PAGE

Form Approved
OMB No. 0704-0188

Public reporting burden for this collection of information is estimated to average 1 hour per response, including the time for reviewing instructions, searching existing data sources, gathering and maintaining the data needed, and completing and reviewing the collection of information. Send comments regarding this burden estimate or any other aspect of this collection of information, including suggestions for reducing this burden, to Washington Headquarters Services, Directorate for Information Operations and Reports, 1215 Jefferson Davis Highway, Suite 1204, Arlington, VA 22202-4302, and to the Office of Management and Budget, Paperwork Reduction Project (0704-0188), Washington, DC 20503.

1. AGENCY USE ONLY (Leave blank)	2. REPORT DATE	3. REPORT TYPE AND DATES COVERED	
	September 1996	FINAL REPORT 05/31/96 -- 08/31/96	
4. TITLE AND SUBTITLE		5. FUNDING NUMBERS	
HEAT TRANSFER IN THIN LIQUID FILMS		C F33615-94-C-3200 PE 62201 PR 2401 TA 02 WU 99	
6. AUTHOR(S)		8. PERFORMING ORGANIZATION REPORT NUMBER	
Rama S. R. Gorla CSA Engineering, Inc		Larry W. Byrd Thermal Structures Branch	
7. PERFORMING ORGANIZATION NAME(S) AND ADDRESS(ES)		9. SPONSORING/MONITORING AGENCY NAME(S) AND ADDRESS(ES)	
CSA ENGINEERING INC. 2850 W BAYSHORE ROAD PALO ALTO CA 94303-3843		THERMAL STRUCTURES BRANCH STRUCTURES DIVISION WRIGHT LABORATORY WPAFB OH 45433-7542	
10. SPONSORING/MONITORING AGENCY REPORT NUMBER		11. SUPPLEMENTARY NOTES	
WL-TM-97-3016		Preliminary Report: Not Approved for Public Release	
12a. DISTRIBUTION AVAILABILITY STATEMENT		12b. DISTRIBUTION CODE	
APPROVED FOR PUBLIC RELEASE; DISTRIBUTION UNLIMITED			
13. ABSTRACT (Maximum 200 words)		<p>The objective of this summer research was to examine heat transfer in thin liquid films. A successful formulation was accomplished on the effect of electrostatic field on the stability of isothermal non-evaporating thin films and on the heat transfer in evaporating thin films. Here, the coupling of the electrostatic field with the fluid dynamics was accomplished through the interfacial boundary condition only. The weakness of this approach lies in the fact that the body forces due to the electrical field are ignored in the equation of motion and the work terms are not included in the conservation law for energy balance. Lastly, a general formulation including the electric field and Van der Waals forces between a solid surface and the fluid in the Navier Stokes equation was written but not applied to a particular geometry. A thorough literature search reveals that analysis that includes these effects has not been undertaken until now. The governing equations for the fluid as well as the electrical field are given for the previously mentioned two problems.</p>	
14. SUBJECT TERMS		15. NUMBER OF PAGES	
Heat Transfer, Thin Liquid Films, Electrostatic Fields, Non-Evaporating Thin Films, Evaporating Thin Films		16	
		16. PRICE CODE	
17. SECURITY CLASSIFICATION OF REPORT	18. SECURITY CLASSIFICATION OF THIS PAGE	19. SECURITY CLASSIFICATION OF ABSTRACT	20. LIMITATION OF ABSTRACT
UNCLASSIFIED	UNCLASSIFIED	UNCLASSIFIED	SAR

TABLE OF CONTENTS

	<u>Page Nr</u>
1.0 INTRODUCTION	1
2.0 EFFECT OF ELECTROSTATIC FIELD ON NON-EVAPORATING FILM RUPTURE	2
3.0 EFFECT OF ELECTROSTATIC FIELD ON EVAPORATING FILMS	7
4.0 EFFECT OF ELECTRICAL FIELD ON EVAPORATING FILMS	10
REFERENCES	12

LIST OF FIGURES

1. FLOW MODEL FOR THE THIN FILM FLOW	2
--------------------------------------	---

FOREWORD

This report was prepared by the Aerospace Structures Information and Analysis Center (ASIAC), which is operated by CSA Engineering, Inc. under contract number F33615-94-C-3200 for the Flight Dynamics Directorate, Wright-Patterson Air Force Base, Ohio. The report presents the work performed under ASIAC Task No. T-29. The work was sponsored by the Structural Integrity Branch, Structures Division, Flight Dynamics Directorate, WPAFB, Ohio. The technical monitor for the task was Dr. Larry W. Byrd of the Structural Integrity Branch. The study was performed by Dr. Rama S. R. Gorla of Cleveland State University, under contract to CSA Engineering Inc.

This technical report covers work accomplished from May 1996 through August 1996.

1.0 INTRODUCTION

The long range objectives of this research are to identify and evaluate the heat transfer characteristics of evaporating thin liquid films.

The optimal design of a heat exchanger is the one that provides the highest heat transfer rate at the lowest overall cost. Derjaguin [1] introduced the concept of the "disjoining pressure" to describe the effect of Van der Waals dispersion forces in very thin films and their role in the dry-out of liquid films. Below a critical film thickness, a non-evaporating absorbed layer exists. As the liquid film increases in thickness, the disjoining pressure gradient enhances flow into the film which they suggested could increase the evaporation rate several fold. Potash and Wayner [2] experimentally determined two distinct regions of the evaporating extended meniscus on a vertical flat plate. Miller [3] studied the stability of liquid-vapor interfaces moving as a result of phase transformation or mass transfer. Das Gupta et al [4] used the Kelvin-Clapeyron change of phase heat transfer model to evaluate experimental data for an evaporating meniscus.

In the present study, attention is focused on the analysis of the following aspects of thin liquid films:

1. The effect of an electric field on the stability of a non-evaporating thin film.
2. The effect an of electric field on evaporating thin films.

The formulation of some of these problems will be summarized in this report.

2.0 EFFECT OF ELECTROSTATIC FIELD ON NON-EVAPORATING FILM RUPTURE

The behavior of a liquid film as it flows down an inclined plane was studied in references [5-10]. All these research workers ignored the London Van der Waals attractions between the solid surface and the liquid and therefore their results cannot be directly applied to the case of thin liquid films.

Our aim here is to address the question of how the thin liquid film and an electrostatic field interact. We are interested in the specific working regimes of the parameters, where it will be possible to stop a rupture (or dryout) of the film. This will be done by solving the equations of film motion in the presence of an electric field. Questions concerning the film stability will be addressed.

We consider the flow of a thin liquid film down an inclined plane under gravity. The plane is assumed to make an angle β with the horizontal. We choose x and y directions to be parallel and normal to the plane, respectively as shown in Figure 1. We assume that the characteristic thickness of the film to be d and the length scale parallel to the film to be L . The aspect ratio $\xi = d/L$. For a thin film, $\xi \ll 1$. The distance from the charged foil and the plane is H .

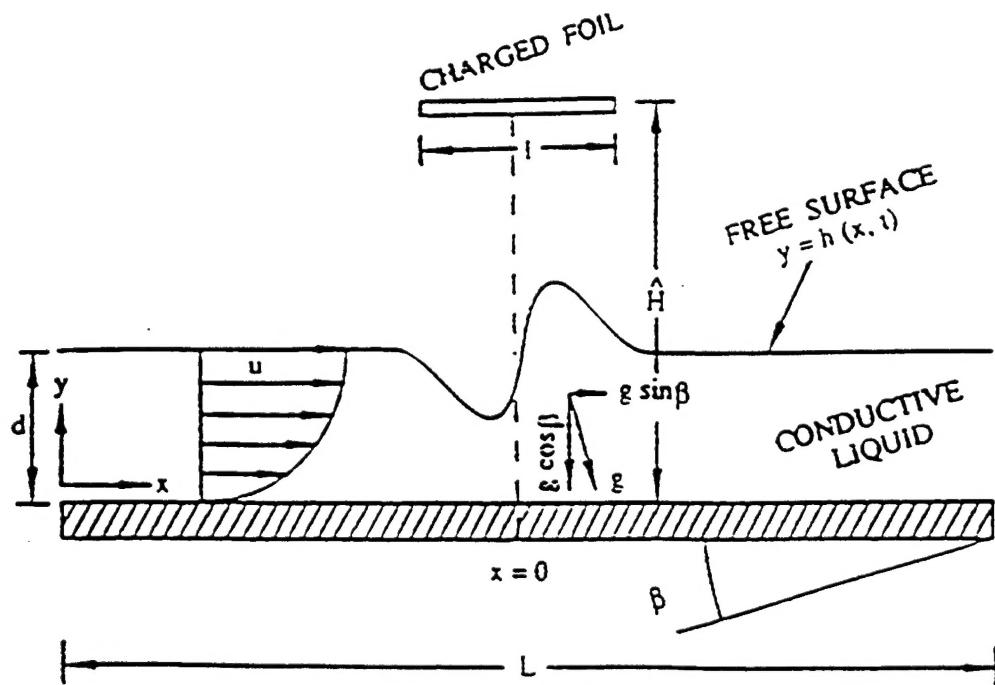


Figure 1. Flow Model for the Thin Film Flow

The electric field is determined by solving Laplace's equation.

$$\nabla^2 \phi = 0 \quad (1)$$

where $\phi(x, y)$ is the electric potential. The boundary conditions are

$$\phi(x, H) = \Phi(x); \phi(x, 0) = 0 \quad (2)$$

Along $y = h(x, t)$ we have the boundary conditions that the tangential electric field and the normal displacement field are continuous. It may be noted that $y = h(x, t)$ is unknown, so that solution of the electrostatic problem is coupled to the dynamics of the film.

The liquid film is governed by the Navier-Stokes equations. The liquid layer is assumed thin enough that Van der Waals forces are effective and thick enough that a continuum theory of the liquid is applicable.

We assume that the liquid is incompressible. The governing equations and boundary conditions are made dimensionless by using the following scales: length in y -direction $\approx d$, in x -direction $\approx L$, velocity in x -direction $\approx U_0$, velocity in y -direction $\approx \xi U_0$, unit of time $\approx L/U_0$, unit of pressure ρU_0^2 , and unit of electric field $\approx F$. We take ρ as the fluid density, ϵ_0 the dielectric constant and μ the viscosity. The continuity equation becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (3)$$

The momentum equation becomes

$$\xi \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\xi \frac{\partial p}{\partial x} + \frac{1}{Re} \left(\xi^2 \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) + \frac{1}{Fr^2} \sin \beta - \xi \frac{\partial \psi}{\partial x} \quad (4)$$

$$\xi^2 \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + \xi v \frac{\partial v}{\partial y} \right) = -\frac{\partial p}{\partial y} + \frac{\xi}{Re} \left(\xi^2 \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) - \frac{1}{Fr^2} \cos \beta - \xi \frac{\partial \psi}{\partial y} \quad (5)$$

In the above equations, u and v are the velocity components in x and y directions respectively, p is the pressure and ψ is the dimensionless potential function representing the van der Waal's forces. We follow Williams and Davis [10] and write a modified expression for ψ :

$$\psi = Ah^{-N} \quad (6)$$

where A is related to the Hamaker constant A' as $A = \frac{A'}{6\pi\rho U_0^2}$. We have introduced the Reynolds number, $Re = \rho U_0 d / \mu$ and the Froude number, $Fr = U_0 / \sqrt{gd}$.

The boundary conditions along the solid plane wall are given by:

$$y = 0; u = v = 0 \quad (7)$$

At the fluid interface, we have the kinematic condition:

$$y = h(x, t) : \frac{\partial h}{\partial t} + u \frac{\partial h}{\partial x} = v \quad (8)$$

The continuity of tangential stress on the interface requires

$$y = h(x, t) : \left[1 - \xi^2 \left(\frac{\partial h}{\partial x} \right)^2 \right] \left(\frac{\partial u}{\partial y} + \xi^2 \frac{\partial v}{\partial x} \right) + 2\xi^2 \frac{\partial h}{\partial x} \left(\frac{\partial v}{\partial y} - \frac{\partial u}{\partial x} \right) = 0 \quad (9)$$

The continuity of normal stress at the interface $y = h(x, t)$ becomes

$$\begin{aligned} \frac{\xi^2}{Ca} \frac{\partial^2 h}{\partial x^2} \left[1 + \xi^2 \left(\frac{\partial h}{\partial x} \right)^2 \right]^{-3/2} + Ah^{-N} &= -\frac{Re}{2} p + K \left(\frac{1}{\epsilon_f} - 1 \right) [(E_n^v)^2 \\ + \epsilon_f (E_t^v)^2] + \xi \left[\xi^2 \left(\frac{\partial h}{\partial x} \right)^2 \frac{\partial u}{\partial x} - \frac{\partial h}{\partial x} \left(\frac{\partial u}{\partial y} + \xi^2 \frac{\partial v}{\partial x} \right) + \frac{\partial v}{\partial y} \right] \times \left[1 + \xi^2 \left(\frac{\partial h}{\partial x} \right)^2 \right]^{-1} \end{aligned} \quad (10)$$

$$K = \frac{\epsilon_0 dF^2}{16\pi\mu U_0}, \text{ where:}$$

ϵ_f = dielectric constant of the fluid, dielectric constant for vapor assumed = 1

ϵ_0 = electrical permittivity of free space

$E_{n,t}^v$ = normal and tangential components of the electric field in the vapor at the interface

Ca = $2\mu U_0/\sigma$ is the capillary number

The left hand side of Eq. (10) includes the term Ah^{-N} to describe disjoining pressure effects explicitly in the interfacial boundary condition. There is still some question about whether this should be included. Williams and Davis [10] did not include this term in a similar analysis. This does not change the leading order values for u_I, v_I, p_I , etc. Equations (3) - (10) determine the motion of the liquid film. Our aim here is to solve for the stability of the liquid film while including the effect of Van der Waals forces and an applied electric field.

We now apply the long-wave theory to study the stability problem. When the layer is thinner than a critical value, small disturbances begin to grow. These waves have wavelengths much larger than the mean thickness of the layer. Defining a small parameter κ that is related to wave number of such disturbances, we may rescale the governing equations:

$$X = \kappa x; Y = y; \tau = \kappa t \quad (11)$$

we now assume the following expansions for the flow field:

$$\begin{aligned}
 u &= u_0 + \kappa u_1 + \kappa^2 u_2 + O(\kappa^3) \\
 v &= \kappa [v_0 + \kappa v_1 + \kappa^2 v_2 + O(\kappa^3)] \\
 p &= \frac{1}{\kappa} [p_0 + \kappa p_1 + \kappa^2 p_2 + O(\kappa^3)] \\
 \Psi_0 &= \kappa \Psi = 0 \quad (1) \text{ as } \kappa \rightarrow 0
 \end{aligned} \tag{12}$$

Neglecting the variation of p with y and substituting expressions (12) into equations (1) - (10), we get

$$u_0 = \left[-\frac{Re}{Fr^2} \sin \beta + Re \cdot \xi \cdot \left(\frac{\partial p_0}{\partial X} + \frac{\partial \Psi_0}{\partial X} \right) \right] \cdot \left[\frac{Y^2}{2} - hY \right] \tag{13}$$

$$v_0 = -Re \cdot \xi \cdot \left(\frac{\partial^2 p_0}{\partial X^2} + \frac{\partial^2 \Psi_0}{\partial X^2} \right) \left[\frac{Y^3}{6} - \frac{hY^2}{2} \right] + \frac{Re}{2} \left[\frac{-\sin \beta}{Fr^2} + \xi \left(\frac{\partial p_0}{\partial X} + \frac{\partial \Psi_0}{\partial X} \right) \right] \left(\frac{\partial h}{\partial X} \right) Y^2 \tag{14}$$

$$p_0 = \frac{2}{Re} \cdot \left(\bar{K} \left(\frac{1}{\varepsilon_f} - 1 \right) [(E_n^v)^2 + \varepsilon_f (E_t^v)^2] - \frac{\xi^2}{\bar{C}a} \frac{\partial^2 h}{\partial X^2} \right) \tag{15}$$

where $\bar{K} = K \cdot \kappa$; $\bar{C}a = Ca/\kappa^3$.

Similarly, expressions u_1 , v_1 and p_1 may be derived. Since these expressions are very long, they are not reproduced here.

Using equations (13) - (15) we may show that the leading order evolution equation for the film rupture is given by:

$$\begin{aligned}
 \frac{\partial h}{\partial \tau} - \frac{h^2}{2} \{ -\frac{Re}{Fr^2} \sin \beta + Re \cdot \xi \left[\frac{2}{Re} \cdot \left(-\frac{\xi}{\bar{C}a} \cdot \frac{\partial^3 h}{\partial X^3} + \bar{K} \left(\frac{1}{\varepsilon_f} - 1 \right) \frac{d}{dX} [(E_n^v)^2 + \varepsilon_f (E_t^v)^2] \right. \right. \\
 \left. \left. - \frac{\kappa A N}{h^{N+1}} \cdot \frac{\partial h}{\partial X} \right] \} \frac{\partial h}{\partial X} \\
 = \frac{Re \xi h^3}{3} \cdot \left\{ \frac{2}{Re} \cdot \left(-\frac{\xi}{\bar{C}a} \cdot \frac{\partial^4 h}{\partial X^4} + \bar{K} \left(\frac{1}{\varepsilon_f} - 1 \right) \right) [(E_n^v)^2 + \varepsilon_f (E_t^v)^2] \right. \\
 \left. + \frac{\kappa A N}{h^{N+1}} \left[-\frac{\partial^2 h}{\partial X^2} + \frac{(N+1)}{h} \left(\frac{\partial h}{\partial X} \right)^2 \right] \right\} \\
 + \frac{h^2}{2} \cdot \frac{\partial h}{\partial X} \cdot \left\{ -\frac{Re}{Fr^2} \sin \beta + Re \cdot \xi \left[\frac{2}{Re} \cdot \left(-\frac{\xi}{\bar{C}a} \frac{\partial^3 h}{\partial X^3} + \bar{K} \left(\frac{1}{\varepsilon_f} - 1 \right) \frac{d}{dX} [(E_n^v)^2 + \varepsilon_f (E_t^v)^2] \right) \right] \right. \\
 \left. - \frac{\kappa A N}{h^{N+1}} \cdot \frac{\partial h}{\partial X} \right\}
 \end{aligned} \tag{16}$$

subject to initial conditions:

$$h(X, 0) = F(X) \quad (17)$$

Equations (16) and (17) may be solved numerically in order to predict the stability characteristics.

3.0 EFFECT OF ELECTROSTATIC FIELD ON EVAPORATING FILMS

Here, we consider a thin liquid film joining an absorbed film and the meniscus of an evaporating interface under the influence of an electrostatic field. From the previous analysis, to a leading order of magnitude, we have

$$p_l - p_v = \frac{\bar{A}}{\delta^4} - \sigma \frac{d^2 \delta}{dx^2} \left[1 + \left(\frac{d\delta}{dx} \right)^2 \right]^{-3/2} + \frac{\epsilon_0}{8\pi} \left(\frac{1}{\epsilon_f} - 1 \right) [(E_n^v)^2 + \epsilon_f \cdot (E_t^v)^2] \quad (18)$$

In the above equation, p_l is the liquid pressure; p_v the vapor phase pressure; \bar{A} the Hamaker constant (negative for a spreading liquid); δ the film thickness; ϵ_f the fluid dielectric constant and E the electric field.

Using the lubrication approximation for the liquid flow in the thin film we have

$$\mu_l \frac{\partial^2 u}{\partial y^2} = \frac{dp_l}{dx} \quad (19)$$

with boundary conditions given by

$$y = 0: u = 0 \text{ and } y = \delta: \frac{\partial u}{\partial y} = 0: \quad (20)$$

From equations (19) and (20) we may write

$$u = \frac{1}{\mu_l} \cdot \frac{dp_l}{dx} \cdot \left[\frac{y^2}{2} - \delta y \right] \quad (21)$$

The mass flow rate per unit width is given by

$$\Gamma = \rho_l \int_0^\delta u dy = -\frac{\delta^3}{3\mu_l} \frac{dp_l}{dx} \quad (22)$$

Following Wayner [47], the evaporative flux by means of the Kelvin-Claperon equation may be written as

$$\dot{m} = a(T_{lv} - T_v) + b(p_l - p_v) = \frac{1}{\left(1 + \frac{a\Delta h_m}{k} \delta \right)} \cdot [a(T_s - T_v) + b(p_l - p_v)] \cdot \quad (23)$$

where

\dot{m} = evaporative flux

$$a = 2 \sqrt{\frac{M}{2\pi R T_{lv}}} \left(\frac{P_v M \Delta h_m}{R T_v T_{lv}} \right)$$

$$b = 2 \sqrt{\frac{M}{2\pi R T_{lv}}} \left(\frac{V_l P_v}{R T_{lv}} \right)$$

T_{lv} = temperature of liquid-vapor interface

T_v = temperature of the vapor

Δh_m = enthalpy of vaporization per unit mass

k = thermal conductivity of the liquid

T_{lv} is related to the surface temperature by the one-dimensional heat conduction equation

$\frac{k}{\delta} (T_s - T_{lv}) = \dot{m} \Delta h_m$. Evaporative flux is related to the flow rate in the film through mass balance in the following form:

$$\frac{d\Gamma}{dx} = -\dot{m} \quad (24)$$

From equations (23) and (24), we may write

$$\frac{1}{3v_l} \frac{d}{dx} \left(\delta^3 \frac{dp_l}{dx} \right) = \frac{1}{1 + \frac{a \Delta h_m}{k} \delta} [a \Delta T + b (p_l - p_v)] \quad (25)$$

where $\Delta T = T_s - T_v$.

we define the following dimensionless variables:

$$\delta_0^3 = -\frac{\bar{A}b}{a \Delta T} = \text{reference thickness}$$

$$\pi_0 = \frac{a \Delta T}{b} = \text{reference pressure}$$

$$\phi = \frac{p_l - p_v}{\pi_0} = \text{dimensionless pressure difference}$$

$$l = \sqrt{\frac{-\bar{A}}{v_i a \Delta T}} = \text{scale factor for } X$$

$$\xi = \frac{x}{l} = \text{dimensionless position}$$

$$\eta = \frac{\delta}{\delta_0}$$

$$\kappa = \frac{a \Delta h m \delta_0}{k} = \text{dimensionless thickness}$$

$$\epsilon = \frac{\sigma \delta_0 b v_i}{(\bar{A})}$$

Dividing equation (18) by π_0 we get the dimensionless form:

$$\phi = -\frac{1}{\eta^N} - \epsilon \frac{d\eta^2}{d\xi^2} + \mathfrak{I}(\xi) \quad (26)$$

$$\text{where } \mathfrak{I}(\xi) = \left(\left(\frac{\epsilon_0 F^2}{8\pi} \right) \cdot \frac{1}{\pi_0} \cdot \left(\frac{1}{\epsilon_f} - 1 \right) \right) [(\bar{E}_n^v)^2 + \epsilon_f (\bar{E}_t^v)^2]$$

$$\text{where } \bar{E}_n^v = E_n^v/F, \bar{E}_t^v = E_t^v/F$$

Equation (25) reduces to

$$\frac{1}{3} \frac{d}{d\xi} \left(\eta^3 \frac{d\phi}{d\xi} \right) = \frac{1}{(1 + \kappa\eta)} \cdot (1 + \phi) \quad (27)$$

Boundary conditions are:

$$\begin{aligned} \xi \rightarrow \infty : \eta \rightarrow 1; \phi \rightarrow -1 \\ \xi \rightarrow -\infty : \eta \rightarrow \infty; \phi \rightarrow \phi_m \end{aligned} \quad (28)$$

Equations (26) - (28) may be solved to predict the effect of electrostatic field on the evaporating film heat transfer.

4.0 EFFECT OF ELECTRICAL FIELD ON EVAPORATING FILMS

In the previous two cases, it has been assumed that the coupling of the electrostatic problem to the fluid dynamics of the film is only through the boundary condition imposed at the liquid-vapor interface. Although this may be valid in the case of weak electrical fields, the case involving strong electrical fields (which is of practical interest) should consider the proper body forces in the momentum equation and the work terms in the energy equation.

References [11-20] consider the effect of electrical fields on single phase flows. A thorough literature search has indicated that the effect of electrical fields on evaporating films or two phase flows has not been investigated so far.

At the conclusion of the present task, there was not enough time to formulate this problem in detail. Therefore, only the governing equations for the problem are given here.

We assume that the fluid is incompressible. The current in the fluid is assumed to obey Ohm's law with the conductivity a function of temperature. The governing equations for the convective heat transfer in an electric field may be written as:

Mass:

$$\nabla \cdot \vec{V} = 0 \quad (29)$$

Momentum:

$$\rho \left[\frac{\partial \vec{V}}{\partial t} + \vec{V} \cdot \nabla (\vec{V}) \right] = - \nabla P + \mu \nabla^2 \vec{V} + q \vec{E} - \frac{E^2}{2} \nabla \epsilon + \vec{F}_{vdw} \quad (30)$$

where \vec{F}_{vdw} is the volumetric body force due to long range intermolecular (Van der Wall) forces.

Energy:

$$\frac{\partial T}{\partial t} + \vec{V} \cdot \nabla T = \alpha \nabla^2 T + \frac{\sigma E^2}{\rho C_p} \quad (31)$$

where

ρ = density of the fluid

μ = viscosity

q = charge density

\vec{E} = electric field

\vec{V} = velocity field

$$P = p_l - \frac{E^2}{2} \rho \left(\frac{\partial \epsilon}{\partial \rho} \right)_T$$

p_l = liquid pressure

In a poorly conducting liquid, the currents are small and therefore magnetic fields are negligible. The equations governing the electric field are given by:

$$\vec{q} = \nabla \cdot (\epsilon \vec{E}) = \text{free charge density}$$

$$\nabla \times \vec{E} = 0$$

$$\frac{\partial \vec{q}}{\partial t} + \nabla \cdot \vec{J} = 0$$

$$\vec{J} = \sigma \vec{E} + \vec{q} \vec{V}$$

where

ϵ = dielectric constant

\vec{J} = current density

σ = electrical conductivity

The boundary conditions are no slip at the solid surface and the same as previously described at the liquid vapor interface.

REFERENCES

1. Derjaguin, B. V. and Zorin, Z. M., "Optical Study of the Absorption and Surface Condensation of Vapors in the Vicinity of Saturation on a Smooth Surface," Proc. 2nd International Congress on Surface Activity (London), Vol. 2, 1957, pp. 145-152.
2. Potash, M. and Wayner, P. C., "Evaporation from a Two-Dimensional Extended Meniscus," Int. J. Heat Mass Transfer, Vol. 15, 1972, pp. 1851-1863.
3. Miller, C. A.; "Stability of Moving Surfaces in Fluid Systems with Heat and Mass Transport," AIChE Journal, Vol. 19, 1973, pp. 909-915.
4. DasGupta, S., Kim, I. Y. and Wayner, P. C., "Use of Kelvin-Clapeyron Equation to Model an Evaporating Curved Microfilm," J. Heat Transfer, Trans. ASME, Vol. 116, 1994, pp. 1007-1015.
5. Benjamin, T. B., "Wave Formation in Laminar Flow Down an Inclined Plane," Journal of Fluid Mechanics, Vol. 2, 1957, pp. 554-574.
6. Yih, C. S., "Stability of Liquid Flow Data on Inclined Plane," Physics of Fluids, Vol. 5, 1963, pp. 321-334.
7. Shen, M. C., et. al., "Surface Waves on Viscous Magnetic Fluid Flow Down an Inclined Plane," Proc. National Heat Transfer Conference, 110, 1989, pp. 161-165.
8. Rahman, M. M., et.al., "Heat Transfer to a Thin Liquid Film with a Free Surface," Proc. National Heat Transfer Conference, 110, 1989, pp. 161-165.
9. Kim, H. and Bankoff, S. G., "The Effect of an Electrostatic Field on Film Flow Down an Inclined Plane," Physics of Fluids, Vol. A4, 1992, pp. 2117-2130.
10. Williams, M. B. and Davis, S. H., "Nonlinear Theory of Film Rupture," Journal of Colloid and Interface Science, Vol. 90, 1982, pp. 220-228.
11. Lazrenko, B. R., Grosu, F. P. and Bologa, M. K., "Convective Heat Transfer Enhancement by Electric Fields," Int. J. Heat & Mass Transfer, Vol. 18, 1975, pp. 1433-1440.
12. Jones, T. B., "Electrohydrodynamically Enhanced Heat Transfer in Liquids - A Review," Advances in Heat Transfer, Vol. 14, 1978, pp. 107-148.
13. Turnbull, R. J., "Effect of Non-Uniform Alternating Electric Field on the Thermal Boundary Layer Near a Heated Vertical Plate," J. Fluid Mechanics, Vol. 49, 1971, pp. 693-703.
14. Turnbull, R. J., "Instability of a Thermal Boundary Layer in a Constant Electric Field," J. Fluid Mechanics, Vol. 47, 1971, pp. 231-239.

15. Berghmans, J., "Electrostatic Fields and the Maximum Heat Flux," *Int. J. Heat & Mass Transfer*, Vol. 19, 1976, pp. 791-797.
16. Maekawa, T., Abe, K. and Tanasawa, I., "Onset of Natural Convection Under an Electric Field," *Int. J. Heat & Mass Transfer*, Vol. 35, 1992, pp. 613-621.
17. Ould El Moctar, A., Peerhossaini, H., Le Peurian, P. and Bardon, J. P., "Ohmic Heating of Complex Fluids," *Int. J. Heat & Mass Transfer*, Vol. 36, 1993, pp. 3143-3152.
18. Lykoudis, P. S. and Yu, C. P., "The Influence of Electrostrictive Forces in Natural Thermal Convection," *Int. Heat & Mass Transfer*, Vol. 6, 1963, pp. 853-862.
19. Barletta, A., "The Temperature Field in a Cylindrical Electric Conductor with Annular Section," *Int. J. Heat & Mass Transfer*, Vol. 38, 1995, pp. 2821-2832.
20. Takano, K., Tanasawa, I. and Nishio, S., "Active Enhancement of Evaporation of a Liquid Drop on a Hot Solid Surface Using a Static Electric Field," *Int. J. Heat & Mass Transfer*, Vol. 37, 1994, pp. 65-71.